

Failure Analysis of Automatic Access Pedestrian Gate Turnstile using SolidWorks Simulation Model

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Abstract- This research work is focused on the failure analysis of Automatic Access Pedestrian Gate Turnstile using SolidWorks Simulation Model. Failure analysis was carried out on both static and dynamic failure of the system putting into consideration the yield stress, resultant displacement, failure cycle, percentage damage, maximum strain and most importantly factor of safety. The results of Von Mises stress analysis showed that a maximum value of 5.77Mpa yield strength was obtained. A displacement of 0.257737m and a strain value of 2.03989×10^{-5} were obtained respectively. The fatigue simulation damage factor was not up to 100% though is up to 50%. At the endurance limit or fatigue limit, the SN curve flattens at 10^6 cycle. A safety of 61 was recorded with the system which is above 1, thus the system is safe and efficient and can be used for security check.

Index Terms- Security, Automatic Access Pedestrian Gate Turnstile, SolidWorks Simulation Model, Failure Analysis

I. INTRODUCTION

Nigeria has consistently ranked low in the Global Peace Index (Azazi, 2011), signifying a worsened state of insecurity in the country. Hence, Adagba et al [1], Uhummwuango and Aluforo [2] are of the opinion that the efforts of government have not yield enough positive result. Security as an essential concept is commonly associated with the alleviation of threats to cherished values, especially the survival of individuals, groups or objects in the near future [3]. As the world is going through a period of technological advancement, the need arises for automated system which can help in security check and the same time operate with little or no human interference. One of such system is the automatic access control system (automated turnstile). Access control system is designed for area where need for security is crucial especially in the institutions. It protects human life, property, information and the same time provides safe and comfortable environment by limiting unnecessary access to essential areas such as offices, building, schools, factories etc. [4,5,6]. It is a biometric access control system that work by identifying the smart card, and in so doing, it prevents any possible danger or interference from the outside and thereby providing safety in any environment [7,8].

Any mechanical system is expected to fail with time. This makes it necessary to carry out failure analysis in design stage and after fabrication/construction. The failure mode could be a static or dynamic failure depending on the operating system. Static failure could be as a results of point load while dynamic failure might be as the consequence of fatigue failure resulting from continuous cyclic stress. Analysis of failure in a mechanical system is a difficult task especially if it is done manually, thus the need for CAD analysis such as SolidWorks Simulation. The Computer-Aided-Design (CAD) is a powerful technology tool that enables researchers to simulate, identify and address critical design failure. In this research work, SolidWorks Simulation Model will be used to investigate all possible failures in the frame assembly of the Automatic Access Pedestrian Gate Turnstile. Both static and dynamic failure analysis will be

carried out to see if the design Automatic Access Pedestrian Gate Turnstile is good enough to be use as a means of security check.

II. METHODOLOGY

SolidWorks is a solid modeler that make use of parametric (i.e. constraints whose values can determine the geometry of the model) features (i.e. building blocks of the part) base approach to create models. Both static and fatigue failure will be analyzed using SolidWorks Simulation model. The fatigue analysis will examine how continuous passages of people through the Automatic Access Pedestrian Gate Turnstile will cause failure. This will allow for proper design for strength and design for life of the system. Figure 2.1 shows the complete model of the simulated turnstile frame assembly, while Table 2.1 shows the solid bodies used for the analysis. In this research work, both dynamic and static failure will be analyzed. For the latter, Von Mises failure criteria will be applied. According to Von Mises failure criteria, yielding of a ductile material commences when the second deviatoric stress invariants reaches a critical value. In other words, failure will occur when the energy of distortion reaches the same energy for yield/failure in uniaxial tension. For the former, fatigue failure of the material will be analyzed and this will be achieved with the SN curve. The SN curve will be used to analyzed failure caused by fatigue. An SN curve defines the number of cycles to failure, when a material is repeatedly cycled through a given stress range.



Figure 1: Complete model

Table 2.1: Solid bodies

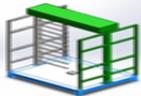
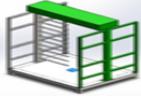
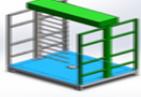
Document Name and Reference	Treated As	Volumetric Properties
Steel portion 	Solid Body	Mass:72.1802 kg Volume:0.00913674 m ³ Density:7900 kg/m ³ Weight:707.366 N
Boss-Extrude5 	Solid Body	Mass:4.11756 kg Volume:0.000521211 m ³ Density:7900 kg/m ³ Weight:40.3521 N
Wooden Portion 	Solid Body	Mass:73.7084 kg Volume:0.00933017 m ³ Density:7900 kg/m ³ Weight:722.342 N

Table 2.2 shows the study properties and material properties is shown in Table 2.3. In this research work, static and dynamic failure are the focus point. Under fatigue stress, mean stress correction with reference to Goodman line will be address. All failures cause by fatigue will be interpreted with SN curve. On static failure, the yield strength of the material will be look into putting thermal effect and every other factors into consideration. Von Mises criteria will be used for confirmation of prospective failure.

Table 2.2: Study properties

Study name	Static 1
Analysis type	Static
Mesh type	Solid Mesh
Thermal Effect:	On
Thermal option	Include temperature loads
Zero strain temperature	298 Kelvin
Include fluid pressure effects from SolidWorks Flow Simulation	Off
Solver type	FFEPlus
Inplane Effect:	Off
Soft Spring:	Off
Inertial Relief:	Off
Incompatible bonding options	Automatic
Large displacement	Off
Compute free body forces	On
Friction	Off
Use Adaptive Method:	Off

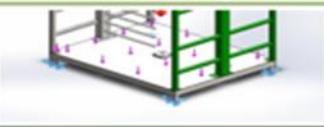
Study name	Fatigue 1
Analysis type	Fatigue(Constant Amplitude)
Event Interaction	Random
Computing alternating stress using	Equivalent stress (von Mises)
Shell face	Top Face
Mean stress correction	Goodman
Fatigue strength reduction factor	1
Infinite life	Off
Result folder	SolidWorks document (C:\Users\ERHINYODAWWE\ONORIODE\Documents\University of Benin\PG Students\Turnstile Project Assembly)

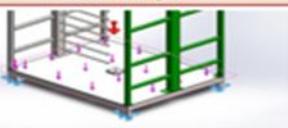
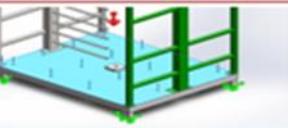
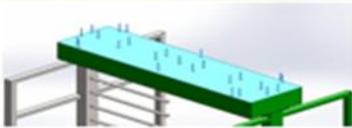
The materials used for the design of the turnstile frame work has the properties shown below. The material has a Poisson's ratio of 0.29. The yield strength of the material is $3.51571e+008$ N/m², while the tensile strength is $4.20507e+008$ N/m².

Name: AISI 1020
 Model type: Linear Elastic Isotropic
 Default failure criterion: Max von Mises Stress
 Yield strength: $3.51571e+008$ N/m²
 Tensile strength: $4.20507e+008$ N/m²
 Elastic modulus: $2e+011$ N/m²
 Poisson's ratio: 0.29
 Mass density: 7900 kg/m³
 Shear modulus: $7.7e+010$ N/m²
 Thermal expansion coefficient: $1.5e-005$ /Kelvin

Table 2.3 shows loads and fixtures information while Table 2.4 shows contact information. In Table 2.3, there are three component of force. These forces are in the X, Y, and Z axis. The resultant forces and moment of the three reactant component forces will determine, and this will help to predict possible failure.

Table 2.3: Loads and fixtures information

Fixture name	Fixture Image	Fixture Details			
Fixed-1		Entities: 4 face(s) Type: Fixed Geometry			
Resultant Forces					
Components	X	Y	Z	Resultant	
Reaction force(N)	-3.05176e-005	4349.02	4.57764e-005	4349.02	
Reaction Moment(N.m)	0	0	0	0	

Load name	Load Image	Load Details	
Gravity-1		Reference: Top Plane Values: 0 0 -9.81 Units: SI	
Force-1		Entities: 1 face(s) Type: Apply normal force Value: 1177.2 N Phase Angle: 0 Units: Deg	
Force-2		Entities: 1 face(s) Type: Apply normal force Value: 196.2 N Phase Angle: 0 Units: Deg	

As shown in Table 2.4, for contact set 1, there are two faces entities, one component, and an option of compatible mesh for global contact.

Table 2.4: Contact Information

Contact	Contact Properties
Contact Set-1	Type: No Penetration contact pair Entities: 2 face(s) Advanced: Node to surface
Global Contact	Type: Bonded Components: 1 component(s) Options: Compatible mesh

The SolidWorks mesh information shown that curvature based mesh was used with Jacobian four points, 182.811mm maximum element sizes, and 36.5622 mm minimum element size. The mesh quality is high, and there was no remesh failed parts with incompatible mesh. Below is the detail summary of the mesh information.

Total Nodes-80983

Total Elements-19760

Maximum Aspect Ratio-536.61

% of elements with Aspect Ratio < 3-20.3

% of elements with Aspect Ratio > 10-26

% of distorted elements(Jacobian)- 0

III. RESULTS AND DISCUSSION

STATIC FAILURE

The reaction force in the X-components is $-3.05176 \times 10^{-5} \text{N}$, while the reaction force in the Y and Z components are 4349.02N, and $4.57764 \times 10^{-5} \text{N}$ respectively. The resultant force of the three components is 4349.02N. However, the reaction moment in Newton meter for each of the components is zero, this means the resultant moment is zero.

Table 3.1: Resultant forces

Reaction Forces

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N	-3.05176×10^{-5}	4349.02	4.57764×10^{-5}	4349.02

Reaction Moments

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N.m	0	0	0	0

The Von-Mises stress analysis results shown that a maximum yield stress of 5.77 Mpa was obtained (Figure 1). A maximum value of 5.77Mpa yield strength will not cause failure. This is because a failure from the analysis will only occur if a yield stress is minimal compared to the yield strength 351Mpa that causes failure.

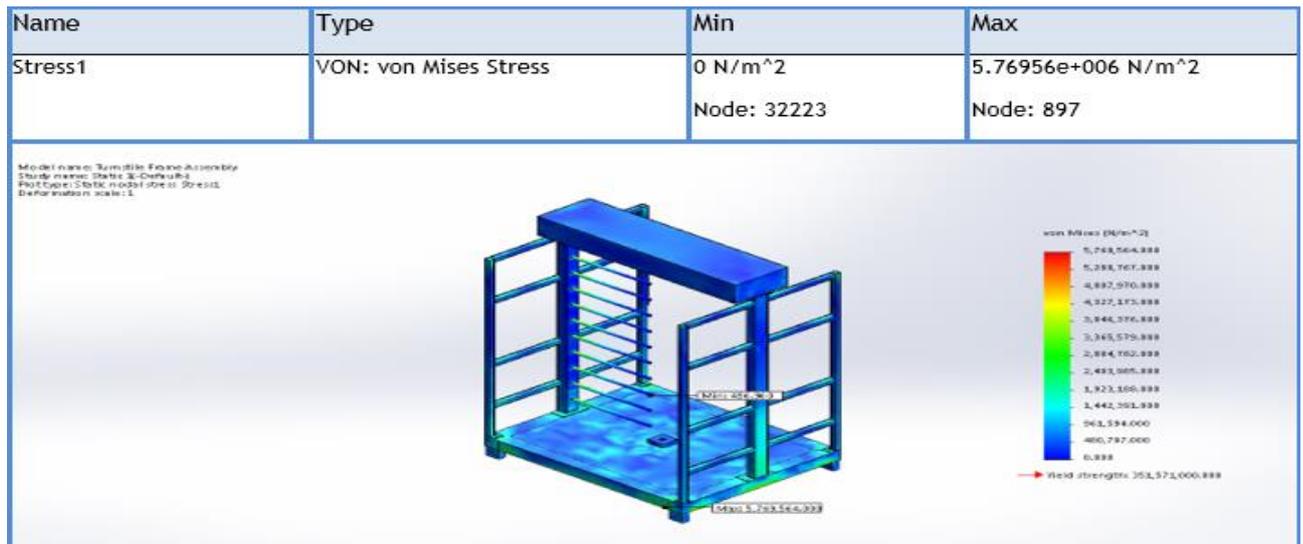


Figure 2: Von- Mises Stress Analysis

The resultant displacement results obtained was 0.257735mm (Figure 2). The displacement of 0.257735mm cause by the effect of the load is small compared to the thickness of the frame support. Therefore, no meaningful failure will result from that.

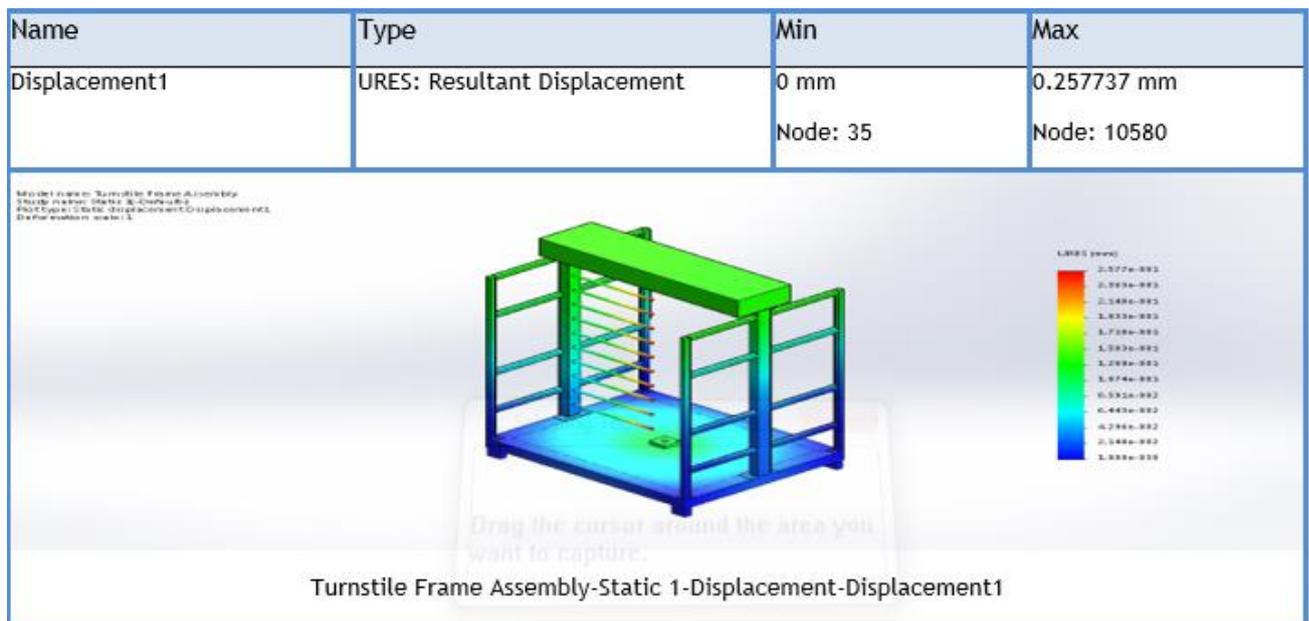


Figure 3: Resultant displacement

From the strain SolidWorks Simulation analysis, a strain value of 2.03989×10^{-5} was obtained and this is small enough not to cause failure (Figure 3).

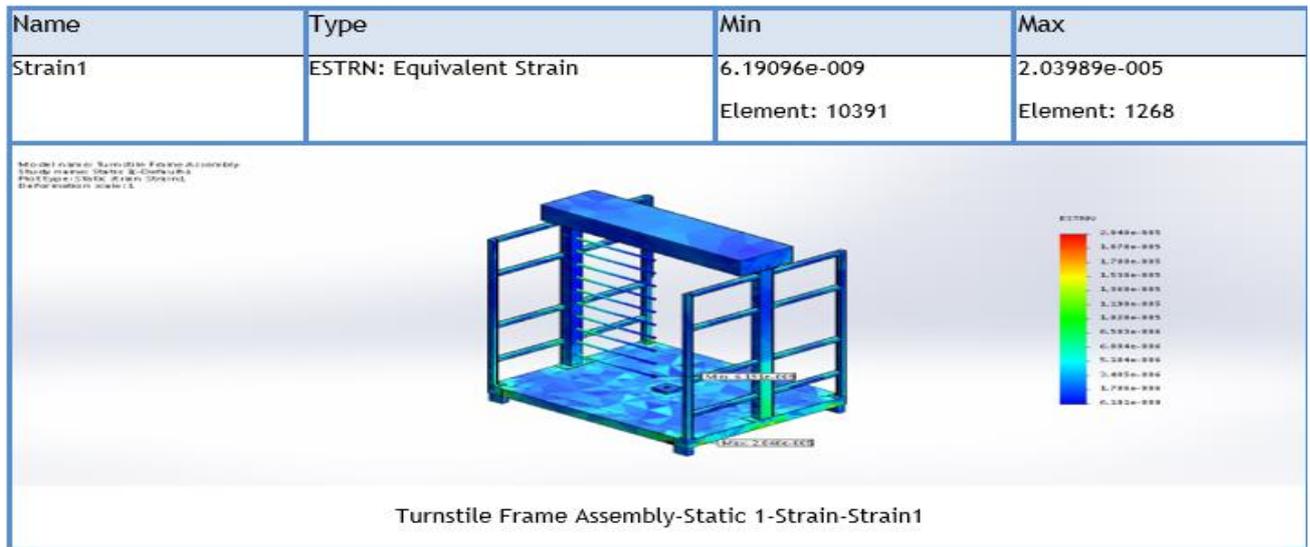


Figure 4: Strain Analysis

FATIGUE FAILURE

The fatigue simulation damage factor is not up to 100% though is up to 50% (Figure 4), therefore, the system is efficient. The system will be unsafe if the damage factor is 100% and above.

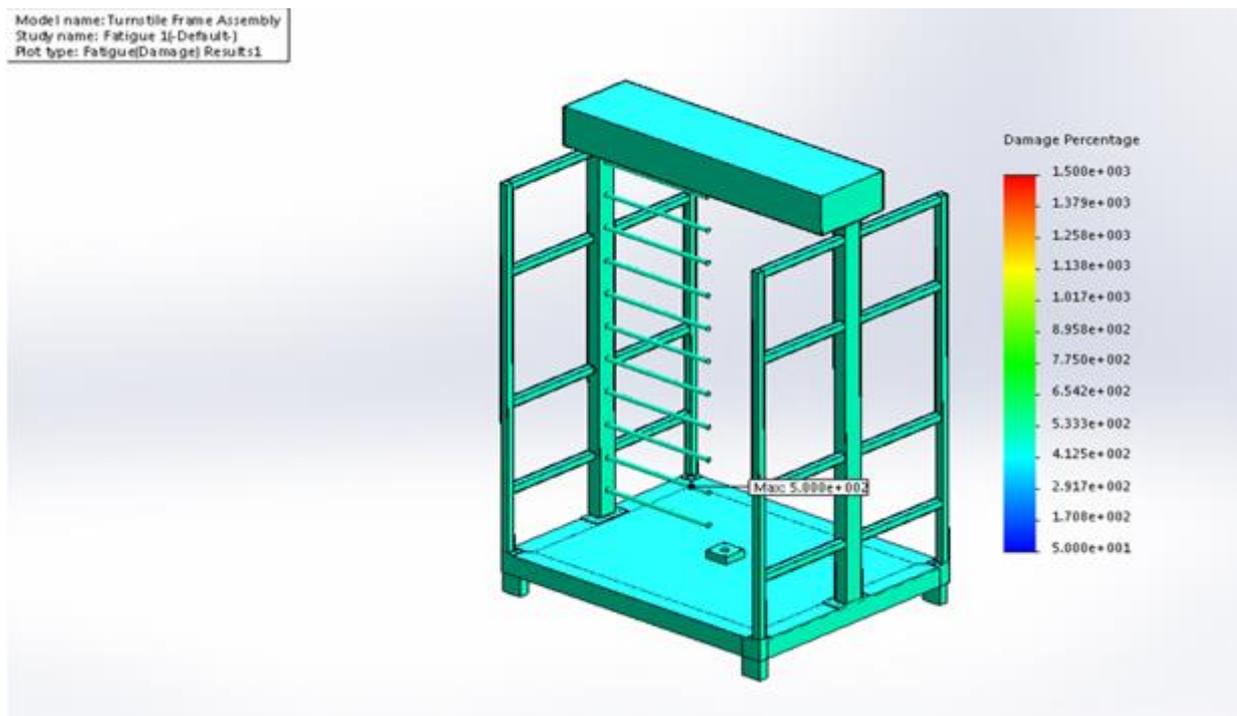


Figure 5: Damage percentage

From the results obtained with fatigue failure analysis (Figure 5), a maximum life cycle of 10^6 was recorded with the system. This implies that the applied continuous stress to the system will not cause fatigue failure.

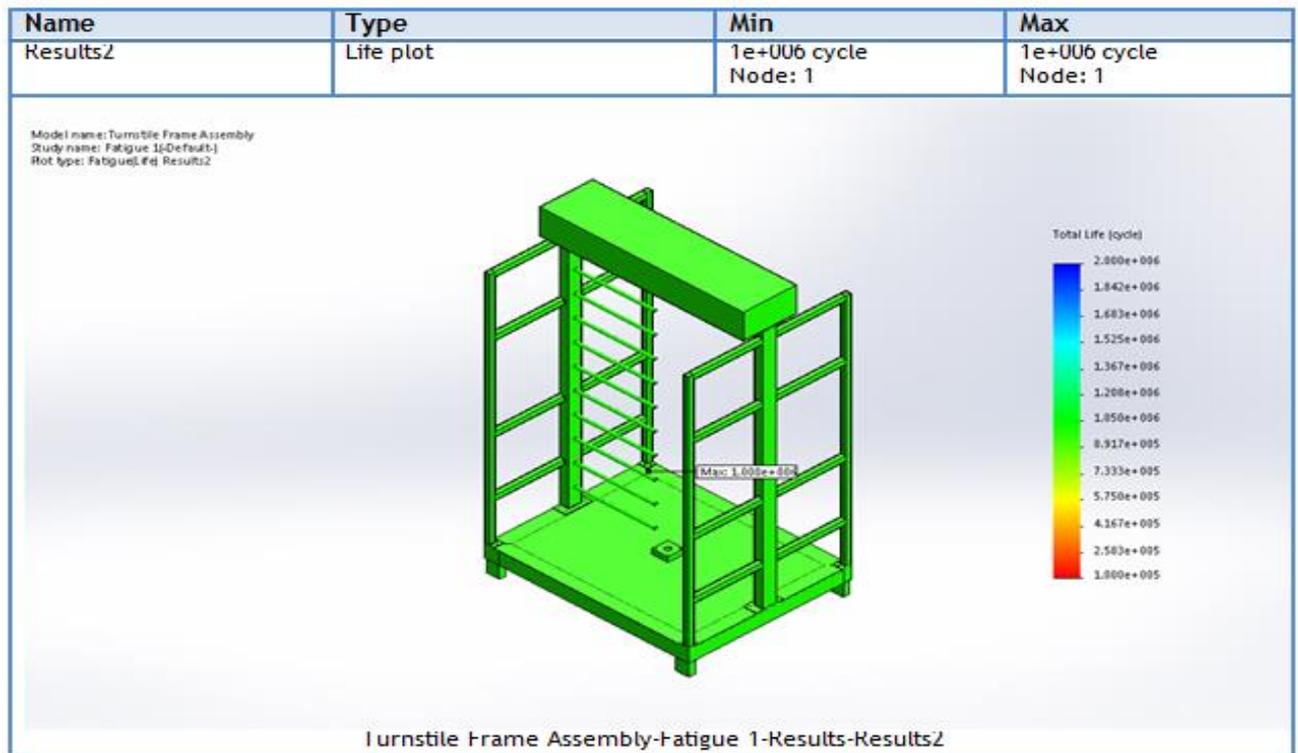


Figure 5: Fatigue Failure Analysis

The SN curve is shown in Figure 6. From the SN curve, the lower the applied stress, the greater the number of cycle to failure. At the endurance limit or fatigue limit, the SN curve flattens at 10^6 cycle. Stress below the fatigue limit can be applied repeatedly in an indefinite number of times without leading to any failure. Thus, the system is safe.

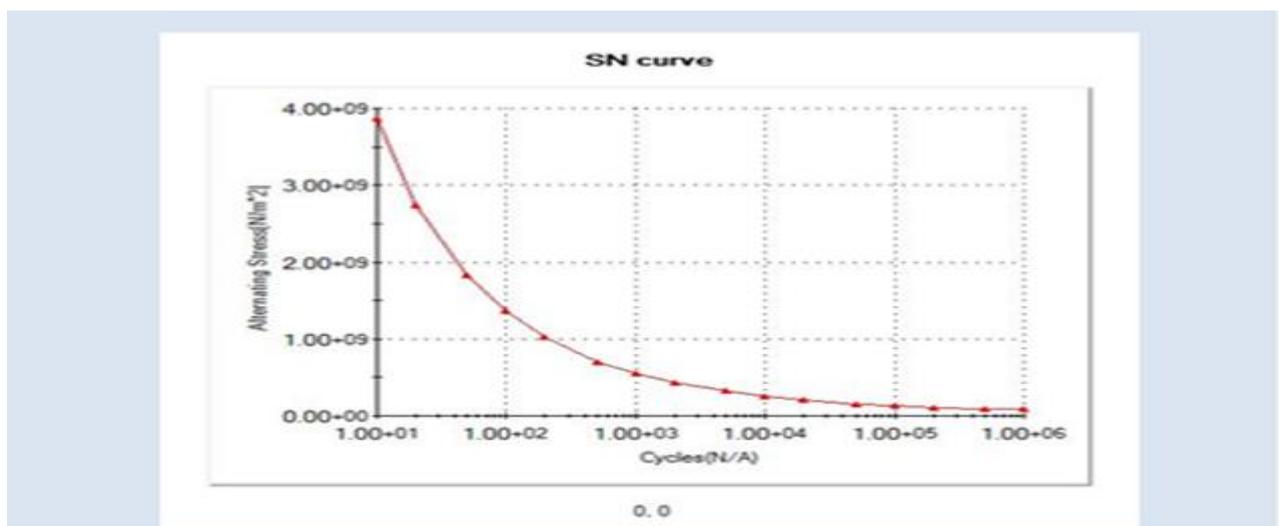


Figure 6: SN Curve

FACTOR OF SAFETY

The factor of safety is 60.9355 (Figure 7). Since the factor of safety is above one (1), the system is safe and efficient.

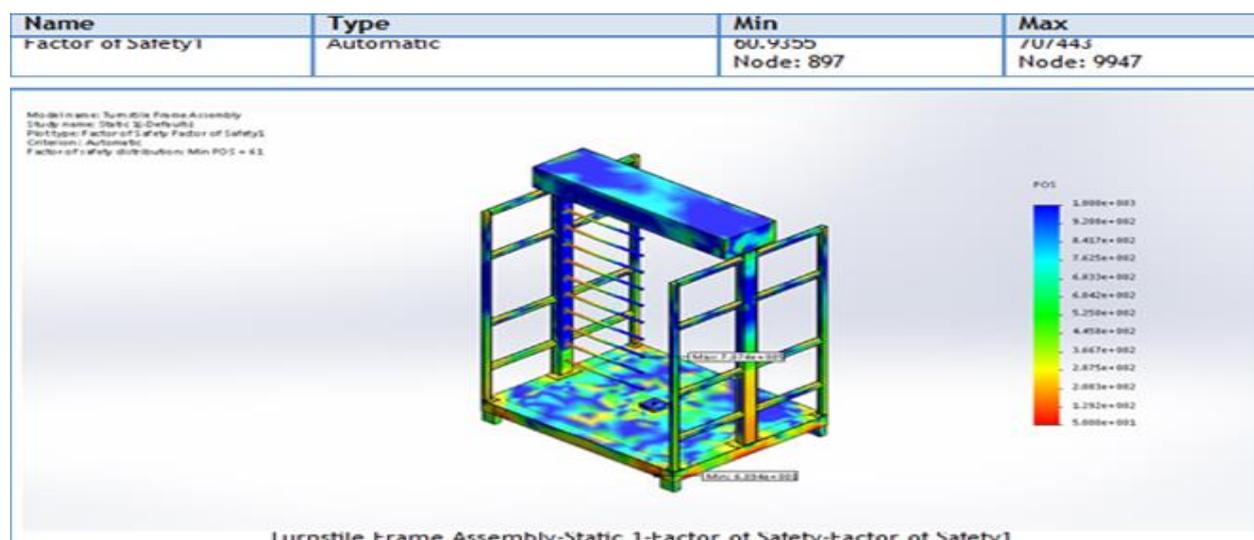


Figure 7: Factor of Safety

IV. CONCLUSIONS

Due to the problem of insecurity especially in high school, college, banks and hospital, a designed pedestrian gate (turnstile) using an ID card and a card reader was analyzed for failure using SolidWorks Simulation Model. The resultants obtained showed that the system can withstand repeated stress without causing failure and a safety of factor above one prove the system to be safe and efficient. Thus, it can be used for security check.

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